

MEASUREMENT METHODS OF INTERNAL STRESS IN CONTINUOUS WELDED RAIL

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ABSTRACT. This paper deals with the problem of internal rail stress estimation. It is based on a detailed research of contemporary situation in the field, presents basic outlines of the problem and sums up the major research areas and their possible applications in the current state of railway infrastructure management.

The directions are divided into four categories in the paper: Displacement Methods, Rail Shifting Methods, Methods Based on Acoustoelastic Effect and Methods Based on Magnetoelastic Effect. Particular methods, both scientific and industrial, are presented in their section respectively.

Every method that is presented within the scope of this paper is briefly described and its advantages and disadvantages are mentioned. In the end, potential of application of some of the presented methods in the practical use is discussed.

KEYWORDS: rail, stress, measurement, CWR, temperature.

1. INTRODUCTION

Continuous welded rail, abbreviated as CWR, is a part of railway superstructure that has been under continuous development since its beginnings in the first half of the twentieth century. The implementation of continuous welded rail caused a major qualitative improvement in the railway transport. One of the biggest challenges of this improvement was the mitigation of the thermal stress effect.

At the moment of welding, there is a certain stress state which equals a temperature called "neutral temperature". The neutral temperature is characterized by the attribute that when a welded rail reaches the neutral temperature, the rail is free from thermal stress. When the rail temperature is higher than the neutral temperature, compressive stress in rail emerges. At extreme temperatures, the compressive stress induced in the rail can cause railway track buckling, also known as "sun kink". On the contrary, when the rail temperature is lower than the neutral temperature, tensile stress in rail emerges. At extreme temperatures, the tensile stress induced in the rail can cause the rail to be pulled apart.

The thermal stress in the continuous welded rail is expressed as

$$\sigma = -\alpha \cdot \Delta t \cdot E \quad (1)$$

where

$$\alpha = 12 \cdot 10^{-6} K^{-1} \quad (2)$$

and

$$E = 210 \cdot 10^9 Pa \quad (3)$$

In the Czech Republic, the permitted welding temperature for rails ranges from 17 °C to 23 °C [1]. Considering the minimum rail temperature at -30 °C and the maximum rail temperature at 60 °C (for rails situated in the sunny locations and thus heated), the difference between the extreme temperature and the neutral temperature can reach as many as 53 °C [2].

Assuming such conditions, the thermal stress in continuous welded rail is

$$\sigma = 133.56 \cdot 10^6 Pa \quad (4)$$

This stress as such is not high enough to cause failure of continuous welded rail. However, there are other contributing factors, too. The thermal stress is cyclic, repeating in both daily and yearly cycles. Trains cause further heavy dynamic loading which is cyclic, too. All these loadings are countable.

It appears, however, that the neutral temperature is not constant over a period of time. Based on the location of the rail, it seems the neutral temperature is shifting. This shift of the neutral temperature can reach even several degrees of Celsius. This difference has to be added to the Δt parameter, eventually increasing the thermal rail stress.

Combinations of the above mentioned stresses and their cyclic character have adverse influence on continuous welded rails. Unlike the other items, it is very difficult to predict the shift of the neutral temperature. This article presents contemporary findings in

1 this area and suggests possible solutions to the evalu-
2 ation of the thermal stress in rail with the influence
3 of the neutral temperature shift.

5 2. DISPLACEMENT METHODS

6 Displacement methods of CWR stress measurement
7 work on the simple mechanical assumption of change
8 of a relative distance between two points of material,
9 rail in this case. In Newton's mechanics, relatively
10 stiff rail behaves according to Hook's law if unfastened.
11 However, even the fastening systems ensure only a
12 certain level of rigidity and, provided the axial force
13 in rail is strong enough, a relative displacement can
14 be observed even at fastened rail sections. As of now,
15 some solutions using displacement are available as
16 presented below.

18 2.1. RESEARCH OF THE GDANSK UNIVERSITY 19 OF TECHNOLOGY

20 This research was carried out in the mid-seventies
21 at the Gdansk University of Technology, Poland. A
22 displacement sensor with accuracy of 0.001 mm has
23 been applied for a measurement on a 200 mm long
24 rail section. Next to the CWR measurement spot, a
25 reference rail section has been placed. This reference
26 rail section has been set free to lengthen and shorten.
27 It was able to observe soon that the stress varies over
28 different sections of the rail even if the temperature
29 of the rail was the same at measured locations.

30 It is believed that this effect is caused by slip of rail
31 in the fastening points. Such a slip can be derived
32 by several sources like tangential forces of traffic, but
33 also by exposure of some parts of rail to sun and
34 location of some parts of rail to completely shady
35 place. Such a different exposure to sun results in
36 different daily mean temperature and, subsequently,
37 to a stress gradient in the rail.

38 This revealing is a good asset of the measurement.
39 The method itself, however, is not suitable for com-
40 mercial use as it is very costly, time demanding and
41 requires a reference rail to be placed at the mea-
42 surement spot. Nevertheless, it can well serve as a
43 reference method for calibration of other methods [3].

46 2.2. THE CALIBRATED LENGTH OF RAIL 47 METHOD

48 This method, which has supposedly been applied in
49 China since 1982, consists of length comparison of a
50 freely placed 50 m long steel strip with a selected rail
51 section originally of the same length. The principle of
52 this method, however, appears to be the same like in
53 the research of the Gdansk University of Technology,
54 only modified for a bigger span. Author's daring claim
55 that this method is the only one in the world without
56 "fatal" shortcomings and can be operated widely does
57 not match the state of the research in this area after
58 the publication date, because much of research has
59 been emerging in this field [4].

61 2.3. MEASUREMENT WITH BI-DIRECTIONAL 62 FBG STRAIN SENSORS

63 One of the most recent papers covering this topic, from
64 2015, presents a method of displacement measurement
65 in a rail web in longitudinal and vertical direction
66 using optic FBG sensors. In general, this measurement
67 resembles strain gauge measurement. However, the
68 advantage of FBG sensors is the possibility to measure
69 even temperature, apart from mere strain. Moreover,
70 it is characterized by its longer durability in relation to
71 strain gauge sensors. Provided the sensor is attached
72 to the rail web prior to CWR installation (i.e. at rail
73 neutral temperature or known rail tension), it keeps
74 measuring the deformation as long as the sensor works.
75 This can be theoretically reached using strain gauges
76 only, although the durability appears as an important
77 factor here. [5].

79 2.4. VORTOK MEASURE AND DETECT

80 Another option to measure displacement can be
81 reached by using the Measure and Detect sensor pro-
82 duced by VORTOK Ltd. company. Simple sensor
83 that is able to measure variety of parameters is made
84 ready to be screwed into a drilled hole in a rail web.
85 The manufacturer does not share any detailed infor-
86 mation of what principles this device uses. It looks,
87 however, like a piezoelectric sensor with accessories
88 that is incorporated into a screw-in dowel.

89 This robust shape most probably predicts a displace-
90 ment method with a long durability. A disadvantage
91 of this method is a necessity of calibration. After cali-
92 bration is made, either by fastening of the device on a
93 stress-free rail or by using another calibration method,
94 the value of internal stress in rail can be obtained in a
95 stable manner. The options to set up automatic data
96 transmission via email or wi-fi even from a moving
97 train make this sensor a user-friendly device [6].

99 3. RAIL SHIFTING METHODS

100 Rail shifting methods of CWR stress measurement
101 use the principle of bending stiffness dependence on
102 axial tension in rail. Provided the rail is in tension,
103 the higher force is necessary to be applied to laterally
104 shift the rail the higher the axial force in the rail is.
105 Apparently, these methods are viable only when the
106 rail is in tension, otherwise failure by buckling may
107 appear.

109 3.1. VERSE METHOD

110 This method has been presented by VORTOK Ltd.
111 company from the United Kingdom. In the first place,
112 the rail shall be released from all fastenings in the
113 length of 30 m. Afterwards, the central part of the
114 unfastened section of the rail shall be lifted into a
115 certain height. The force that is applied to move the
116 rail is related to the axial force in the rail.

117 Wide use of this method is restricted by some dis-
118 advantages. The most obvious one is the limitation to
119 tensile stress in the measured rail. The operator needs
120

1 to be sure that there is a tensile stress as problems
2 with re-installation of the rail into the original posi-
3 tion could occur, leave apart the danger of operator's
4 injury or material damage. Another disadvantage is
5 the need to unfasten the rail in the length of 30 m.
6 This operation requires additional time and subse-
7 quently prolongs track closure. Moreover, in case the
8 measured section is situated in radius, re-installation
9 of the rail can be difficult, too, as the condition of
10 tensile stress in rail is inevitable.

11 Even though the disadvantages are important, this
12 method appears to be one of the most widely spread
13 to measure the CWR stress nowadays. The man-
14 ufacturer produces their own device for rail lifting
15 and stress measurement. Light aluminium design of
16 this device enables easy transport, manipulation and
17 operation [7].

18 3.2. USE OF TAMPING MACHINE

19 In a paper published in 2009, scientists at Gdansk
20 University of Technology look into a possibility to
21 use tamping machine for stress measurement in CWR.
22 Tamping machine is capable to shift rails in both ver-
23 tical and horizontal direction when adjusting railway
24 track geometry. The goal of this research was to use
25 this movement to estimate the stress in CWR. The
26 principle of this measurement is similar like at the
27 VERSE method, including the disadvantages. How-
28 ever, the disturbing signal produced by the operation
29 of tamping machine turned out to be too high and
30 the attempts to get some CWR stress data were un-
31 successful [8].

32 4. METHODS BASED ON 33 ACOUSTOELASTIC EFFECT

34 The acoustoelastic effect is a physical effect of change
35 of sound velocity based on change of mechanical stress
36 in elastic continuum. For measurements, ultrasound
37 is used. The definition, however, can cover vibration,
38 as mechanical waves, too. In such a case, vibrometer
39 is used as a measurement tool.

40 It is always necessary to execute a calibration mea-
41 surement on the site with known temperature and
42 mechanical stress as the sound velocity in rail is not
43 dependent only on mechanical stress, but on steel
44 microstructure, too. This calibration measurement
45 can be performed either prior to CWR installation,
46 or once the CWR is installed. However, in the lat-
47 ter only when the rail is cut, unfastened and welded
48 again. As a consequence, similar disadvantage like at
49 displacement methods emerges, which means the need
50 to start the measurement at the neutral temperature.
51 The advantage, on the contrary, is in possible high
52 durability of ultrasound sensors and their attachment
53 to measurement place.

54 4.1. J. SZELAZEK MEASUREMENT

55 J. Szelazek has worked out two ultrasound measure-
56 ment procedures in the nineties of the twentieth cen-
57 tury. In the first procedure, ultrasound signal is emit-
58 ted vertically from the top of rail head to the bottom
59 of rail foot where it reflects off and moves back to the
60 receiver, which can be the same device as the emit-
61 ter. The second procedure uses separate emitter and
62 receiver and those are placed to the side of rail head
63 horizontally next to each other. The signal moves
64 through the rail head lengthwise, in this case.

65 The first procedure showed as inapplicable, as the
66 signal dispersed due to uneven rail head profile. The
67 major cause of this unevenness is the rail head wear.
68 Placing the signal receiver to the bottom of rail foot
69 did not improved the results and, moreover, the signal
70 path was reduced to one half.

71 The second procedure appeared more promising,
72 but even at this measurement, disturbances appeared
73 and the results were influenced by strong disper-
74 sion [9].

75 4.2. RESEARCH OF THE UNIVERSITY OF 76 FLORENCE

77 In 2007, Italian scientists worked out the second pro-
78 cedure of J. Szelazek. This procedure has been tested
79 for 2 years and appears viable. However, it shares
80 all the common disadvantages of ultrasound methods
81 and the necessity to perform calibration measurement
82 in situ [10].

83 4.3. MEASUREMENT OF STRESSES USING THE 84 POLARIZATION OF RAYLEIGH SURFACE 85 WAVES

86 Two US researches work with measurement of polar-
87 ization of Rayleigh surface waves. Measurement of
88 dependence of the Rayleigh surface waves polarization
89 on change of mechanical rail stress turned out not just
90 to be more robust than measurement of dependence
91 of the Rayleigh surface waves velocity on change of
92 mechanical rail stress, but also to be easier to detect.

93 Michael D. A. Junge's work deals with general mea-
94 surement of mechanical stress in material based on
95 Rayleigh surface waves [11]. Stefan Hurlebaus' report
96 deals with application of this procedure on rail stress
97 research. On page 42, the author presents that the
98 polarization of Rayleigh surface waves is dependent
99 on the value of axial force in rail and that a further
100 research is recommended [12].

101 4.4. VIBRATION MEASUREMENT

102 Two publications, a diploma thesis and a research re-
103 port, deal with vibration measurement of the internal
104 rail stress. Both of them are outputs of one research.
105 The results of the rail stress determination from wave
106 lengths of vibrating sensor seem well in laboratory
107 conditions when a steel bar or a new rail is used. In
108 case of worn rail, and this is very important for prac-
109 tical application, it is possible to get certain results,
110 but these results are not that clear like in the previous
111 cases. When performing a field measurement, the
112 results of measurement on worn rail are much worse.
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As of yet, this method is inapplicable. In case it is possible to diminish the problems, this method can have a good perspective [13, 14].

5. METHODS BASED ON MAGNETOELASTIC EFFECT

The magnetoelastic effect is inverse magnetostrictive effect. It is presented as a change of magnetic susceptibility in relation to change of mechanical stress in material. The magnetoelastic effect is also known as the Villari effect.

5.1. MEASUREMENT OF MAGNETIC HYSTERESIS

In the nineties, D. Utrata tested the possibility of application of the magnetoelastic effect on the rail stress measurement. He measured the dependence of coercivity and remanence of the magnetic field in rail on different surface conditions, like scaled rail surface, or milled rail surface. However, the comparison of the obtained data has shown various results based on the input type of rail. Surprisingly, the only data that match were on the milled running surface and scaled base underside. D. Utrata assumes, in the end, that either a calibration measurement or a new approach to data interpretation has to be delivered in order to make this method applicable.

A. Wegner needed 400 calibration sensors to run a measurement of 3 meters long rail using the hysteresis method, which does not contribute to viability of this method [14, 15].

5.2. MEASUREMENT OF MAGNETIC BARKHAUSEN EFFECT

Japanese scientists Tsuchima and Enokinozo have studied the reaction of the Barkhausen noise on mechanical stress in steel plate. It is possible to get some relation from the graphs of the mutual dependence. However, the authors found it difficult to estimate the influence of time and get a value of the mechanical stress. Additionally, they suggest application of the chaos theory to get a better interpretation of the results [16].

Measurement using the Barkhausen noise together with measurement of magnetic permeability is used by Elektro-Thermit GmbH & Co. KG. company to determine the internal rail stress. Their method requires calibration on a rail test sample. Their product has been awarded a certificate of the railway infrastructure manager of Denmark, Banedanmark. On the contrary, the manufacturer appears not interested in sale of their product and only offers to perform a measurement in the required section by themselves [3, 17].

5.3. METAL MAGNETIC MEMORY MEASUREMENT

Collective of authors from the University of Nanjing have carried out an experiment of measuring the effect called Metal Magnetic Memory on a steel test sample.

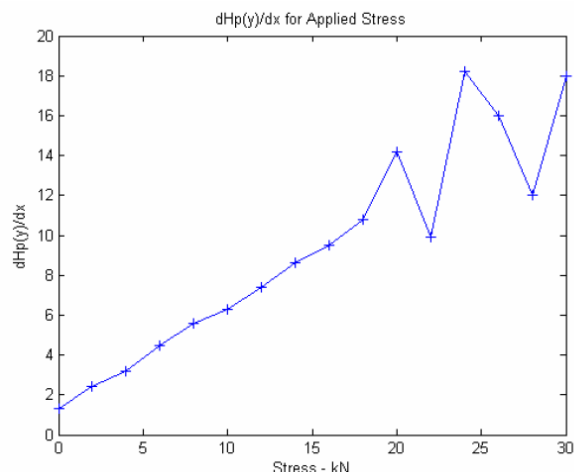


FIGURE 1. $dH_p(y)/dx$ for applied tensile stress on a steel sample [18]; see the linear dependence in the elastic region.

Contrary to the Barkhausen noise, this effect does not require external magnetization of the sample. The effect is activated anywhere in the natural magnetic field of the Earth. When the loading increases, the rotation of the Weiss magnetic domains in metals raises the value of the magnetic flux around the central point and this increases the gradient of the magnetic flux which is in elastic region linearly dependent on the stress in the sample. Graph of the linear dependence of the magnetic flux gradient on the mechanical stress in a steel sample as measured by Wang, et. al. [18] is presented in Figure 1. The linear dependence in the elastic region is completely sufficient, because rail geometry deformation emerges earlier than plastic deformation in the axial direction of the rail [18].

6. DISCUSSION

Every presented method has its limitation. Displacement methods can be related to widely spread strain gauge measurement. This has been used in many industrial application; in this case, however, the biggest problem seems to be the necessity to measure an absolute value of stress, not a relative one, which is the common purpose of strain gauge measurement in general.

Rail shifting methods, especially the VERSE method, seem to be one of the most reliable at this time. The reliability is underpinned by its simplicity. However, more extensive use of this method is very limited through many disadvantages it inevitably drags along. Movement of a rail is a complicated operation, which needs expensive tools and is very time demanding. Re-installation of the rail requires verification that the rail is in good condition after the measurement and safety of operation on the track is kept. Additionally, performing such a measurement in radius is much more complicated and performing such a measurement when the rail is under a compressive stress is excluded. Considering the compressive stress

1 in rail is usually during the day, the limitations of use
2 of this method are vast.

3 Methods based on the acoustoelastic and magne-
4 toelastic effect have the potential to become viable in
5 the non-destructive rail stress measurements. As of
6 yet, however, the problems of getting clear values of
7 stress were not mitigated and residual stresses, diffi-
8 cult conditions of field measurement, or even various
9 conditions of the rail itself (wear, rust, etc.) prevented
10 the research to reach a satisfactory solution.

11 6.1. FURTHER RESEARCH ORIENTATION

12 Out of the contemporary possibilities, there are some
13 ways to be worked out in order to either get satisfac-
14 tory results or to discover serious obstacles that
15 hinder their application.

16 (1.) Application of the Metal Magnetic Memory
17 Method on internal rail stress measurement seem
18 like a research that has not been done yet. As men-
19 tioned in Section 5, this method does not require
20 external magnetization [18] and this is a great asset
21 for possible further application. If the measurement
22 of the gradient of magnetic flux in rail results in
23 a similar dependence as shown in Figure 1, the
24 potential of applicability of this method would be
25 great [18].

26 (2.) Vibration measurement, or modal analysis could
27 also offer an approach to obtain data of internal
28 stress in CWR. However, damping caused by rail fas-
29 tenings could have too high influence on the results.
30 As presented in Section 4, the vibration analysis
31 performed by the team of the University of Illinois
32 at Urbana-Champaign did not produce any applica-
33 ble results even after they released some fastenings,
34 which is a procedure that is disadvantageous as
35 such [13, 14].

36 (3.) Provided the stress could be evaluated from a
37 surface layer of rail only, i. e. there was a mutual
38 relation between surface stress and stress in the
39 core of the rail, another possible method emerges.
40 Method of dependence between passive layers forma-
41 tion in electrolyte and mechanical stress in the
42 surface layers has been shown in an experiment of
43 stainless steel in a normal sulphuric acid bath [19].

44 (4.) Another option is to work out the measurement
45 of stresses using the polarization of Rayleigh surface
46 waves. The papers presented in Section 4 suggest a
47 further research can have a potential to find out a
48 viable solution [11, 12].

49 (5.) Finally, if strain gauge is able to measure a quasi
50 static load change in a long term, a statistical eval-
51 uation of measurements could provide a range of
52 data that could be sufficient for prediction of neu-
53 tral temperature level and subsequently stress level
54 in rail. If such a research is biased towards a certain
55 limited location, an applicable methodology can be
56 a reasonable output.

57 7. CONCLUSIONS

58 Many attempts to measure the neutral temperature or
59 stress in continuous welded rail have been undertaken.
60 Displacement and rail shifting methods like VORTOK
61 Measure and Detect or VERSE have been finalized in
62 an applicable product and offered on market. These
63 methods come out of mechanical solutions. However,
64 these methods suffer from many limitations and re-
65 quirements of special conditions that are necessary to
66 be created in order to use the methods.

67 Later in time, non-destructive methods based on
68 the acoustoelastic or magnetoelastic effects have been
69 tested. As of yet, none of them reached the level of
70 being reliable enough to compete with the above men-
71 tioned ones. Nevertheless, there are some promising
72 results and further research in this area, as described
73 in Section 6, could prove viability of some of them.

74 Provided strain gauges are capable of measuring
75 quasi static load in a long term and a thorough strain
76 gauge measurement is done, the value of stress in a
77 continuous welded rail could be estimated on a certain
78 geographical area even from statistical data measured
79 by strain gauges.

80 Finally, the possible solution can lurk in another
81 method of stress evaluation in a continuum, a method
82 which is not yet known to the authors of this paper.

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